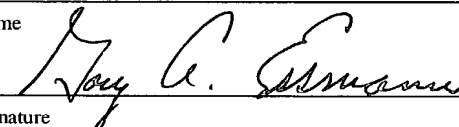


Items 11 to 20 below concern other document(s) or information included:

- 11. ☐ An Information Disclosure Statement under 37 CFR 1.97 and 1.98.
- 12. ☐ An assignment document for recording. A separate cover sheet in compliance with 37 CFR 3.28 and 3.31 is included.
- 13. ☒ A FIRST preliminary amendment.
- 14. ☐ A SECOND or SUBSEQUENT preliminary amendment.
- 15. ☐ A substitute specification.
- 16. ☐ A change of power of attorney and/or address letter.
- 17. ☐ A computer-readable form of the sequence listing in accordance with PCT Rule 13ter.2 and 35 U.S.C. 1.821-1.825.
- 18. ☐ A second copy of the published international application under 35 U.S.C. 154(d)(4).
- 19. ☐ A second copy of the English language translation of the international application under 35 U.S.C. 154(d)(4).
- 20. ☐ Other items or information:
 - ☐ Applicant claims small entity status.
 - ☒ Supplement to Transmittal Letter.

JC13 Rec'd PCT/PTC 01 APR 2002

U.S. APPLICATION NO. (if known, see 37 CFR 1.5) 10/089559	INTERNATIONAL APPLICATION NO. PCT/EP00/09705	ATTORNEY'S DOCKET NUMBER 825-164
CERTIFICATE OF EXPRESS MAIL		
I hereby certify that this correspondence is being deposited with the United States Postal Service with sufficient postage as EXPRESS MAIL-POST OFFICE TO ADDRESSEE, in an envelope addressed to: BOX PCT, COMMISSIONER OF PATENTS AND TRADEMARKS, WASHINGTON, D.C. 20231 on the 1st day of April, 2002. Express Mail Label EV 097314852 US.		
GARY A. ESSMANN	29,376	
Name	Reg. No.	
	April 1, 2002	
Signature	Date	

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

Application of:)
 OTFRIED SCHWARZKOPF) AXIAL PISTON COMPRESSOR)

PRELIMINARY AMENDMENT

Milwaukee, Wisconsin 53202

Box Patent Application
Asst. Commissioner for Patents
Washington, D.C. 20231

Sir:

It is requested that the U.S. national stage examination be carried out on the amended claims dated July 12, 2001. Prior to computing the filing fee in this application, kindly amend the above identified application, as follows. The filing fee is to be computed on the amended claims.

In the Specification:

Beginning at page 1, between the title and the first line of text, the specification has been amended as follows:

CROSS REFERENCE TO RELATED APPLICATION

The present application is the U.S. national stage application of International Application PCT/EP00/09705, filed October 4, 2000, which international application was published on April 12, 2001 as International Publication WO 01/25635 A1. The International Application claims priority of German Patent Application 199 47 677.2, filed October 4, 1999.

In the Claims:

Claim 3 has been amended as follows:

3. Axial piston compressor according to Claim 1, characterized in that the disc is a wobble plate that is rotatably mounted on a swash plate and is set at a tilt angle with respect to the drive shaft that corresponds to the angle of the swash plate.

Claim 4 has been amended as follows:

4. Axial piston compressor according to claim 1,
characterized in that, given a distance of 30 mm between the long axis (L) of the drive shaft and the long axis (Z) of the piston, an 8-mm diameter of the flat surface (22) of the sliding blocks (20), which is opposed to the slideway, and a maximal tilt angle (α) of 18° between the long axis of the drive shaft and the central axis of the disc, the distance between the mid-plane of the disc and the pivotal axis of the disc (14) is no greater than about 1 mm.

Add the following new claims:

5. Axial piston compressor according to Claim 2, characterized in that the disc is a wobble plate that is rotatably mounted on a swash plate and is set at a tilt angle with respect to the drive shaft that corresponds to the angle of the swash plate.

6. Axial piston compressor according to claim 2,
characterized in that, given a distance of 30 mm between the long axis (L) of the drive shaft and the long axis (Z) of the piston, an 8-mm diameter of the flat surface (22) of the sliding blocks (20), which is opposed to the slideway, and a maximal tilt angle (α) of 18° between the long axis of the drive shaft and the central axis of the disc, the distance between the mid-plane of the disc and the pivotal axis of the disc (14) is no greater than about 1 mm.

Respectfully submitted,



Gary A. Essmann
(Reg. No. 29,376)

ANDRUS, SCEALES, STARKE & SAWALL, LLP
100 East Wisconsin Avenue, Suite 1100
Milwaukee, Wisconsin 53202
(414) 271-7590
Atty. Docket No. 825-164

OTFRIED SCHWARZKOPF

Atty. Docket No. 825-164

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Marlene Kubiak

Name

Reg. No.



April 1, 2002

Signature

Date

3/parts

5

Axial piston compressor

10 State of the art

The invention relates to an axial piston compressor with a drive shaft, a disc mounted on the drive shaft so that it can be tilted relative to the latter about a pivotal axis, and at least one piston provided with sliding blocks that move along a slideway on the disc.

Such an axial piston compressor can be used in particular in an air conditioner for motor vehicles. It serves to suck a coolant out of a heat-transfer compartment, in which the coolant evaporates while taking up heat, and to compress it to a higher pressure so that in another heat-transfer compartment the heat can be given off at a higher temperature level. Subsequently the coolant passes into an expansion organ, where it is returned to the pressure level of the first heat-transfer compartment.

For vehicle air conditioners coolant compressors of various constructions are employed. In recent years, for several reasons, axial piston compressors have come into general use, in particular because this construction enables an energetically favourable regulation of the output. That is, the compressor is customarily coupled directly to the motor by a belt drive, so that the operating conditions of the compressor cannot be adjusted as desired by changing the rotational speed of the compressor; for this reason the output is altered by

tilting the disc, which determines the volume displaced by the compressor piston.

The stroke of each piston is produced by the cooperation between the sliding blocks connected to the piston and the disk, which can be pivoted relative to the drive shaft. When the disk is not tilted with respect to the drive shaft, i.e. the central axis of the disc coincides with the long axis of the drive shaft, there is no stroke, because the distance between, for example, the floor of the cylinder within which the piston is disposed and the bearing surface does not change when the drive shaft rotates. On the other hand, when the disc is tilted so that the angle between the central axis of the disc and the long axis of the drive shaft is different from zero, usually at most 20° , the distance between the bearing surface of the disc and the floor of the cylinder changes periodically between a minimal and a maximal value during each rotation of the drive shaft. Thus when the distance is minimal, the piston coupled to the disc is at its top-dead-centre position, i.e. is inserted maximally into the cylinder, whereas when the distance is maximal, the piston is at bottom dead centre.

The slideway, i.e. the path on the disc surface along which the sliding blocks mounted on the piston move, changes according to the angle at which the disc is tilted.

When the central axis of the disc coincides with the long axis of the drive shaft, the sliding blocks move over the disc along a circular slideway, the radius of which corresponds to the distance between the centre of the sliding blocks and the long axis of the drive shaft. In contrast, when the disc is tilted, the sliding blocks move along an elliptical slideway, because the distance between the middle of the sliding blocks and the long axis of the drive shaft is unchanged. The minor axis of the ellipse has a length corresponding to the radius of the circular slideway on a disc that is not tilted, and is parallel

to the pivotal axis of the disc. The length of the major axis of the ellipse is equal to the radius divided by the cosine of the tilt angle of the disc.

To make the compressor compact, the pivotable disc is
 5 dimensioned so that when it is not tilted, there remains only a very small margin between the slideway of the sliding blocks and the outer edge of the disc. As a result, when the disc is tilted, the slideway overlaps the edge of the disc in the regions of the disc that correspond to the upper and the lower
 10 dead-centre points. This is a consequence of the apparent shortening of the disc when it is pivoted. Because of the fact that the slideway overlaps the edge when the disc is tilted, the area available to transfer the forces between disc and sliding blocks is reduced. Furthermore, in one of the positions
 15 in which the sliding blocks overlap the edge of the disc to the greatest extent, namely the position corresponding to the top-dead-centre point of the piston at the end of the compression stroke, the force exerted between the sliding blocks and the disc is maximal. Because the reduction of the area available
 20 for force transfer coincides with the maximum of the force to be transferred, the surface pressure between the disc and the sliding blocks increases, which in the extreme case can cause severe abrasion between these structures.

The objective of the invention is thus to improve an axial
 25 piston compressor of the kind described above in such a way that abrasion between the sliding blocks and the disc is reliably prevented under all operating conditions.

Advantages of the invention

In an axial piston compressor in accordance with the invention,
 30 with the features cited in the characterizing part of Claim 1, the pivotal axis of the disc is offset from the disc's central plane; as a result, a translational movement is superimposed on the rotational movement of the disc. The consequence is that

Preferably it is provided that the displacement of the pivotal axis of the disc from the mid-plane of the disc is towards the side of the disc that faces the piston. In this configuration the reduction of contact area between the sliding blocks and the disc brought about by tilting of the disc is counteracted in the region corresponding to the top-dead-centre point of the associated piston, i.e. at the operating point at which the force acting on the piston is greatest. The reduction of contact area between sliding block and edge of the disc that does occur in this configuration, which is twice as great as in a configuration according to the state of the art (with a pivotal axis that coincides with the mid-plane of the disc), can be tolerated because at the corresponding time the force acting on the piston is comparatively slight. Even though the contact area between sliding blocks and disc surface is reduced, the resulting surface pressure is below the critical values.

25 According to one preferred embodiment of the invention the disc is a swash plate, which can be set into rotation by the drive shaft and the tilt angle of which with respect to the drive shaft can be adjusted. Such an axial piston compressor, which -
30 the rotational movement of the disc - corresponds to the structure known for example from the patent DE 197 03 216 A1, combines the advantage obtained in accordance with the invention, namely a reduction of surface pressure at certain times during operation such as the time when the force acting

According to a preferred embodiment of an axial piston compressor in accordance with the invention it is provided that with a distance of 30 mm between the long axis of the drive shaft and the long axis of the piston, an 8-mm diameter of the flat surfaces of the sliding blocks, which are apposed to the disc, and an angle of maximally 18° between the long axis of the drive shaft and the central axis of the disc, the distance between the mid-plane of the disc and the pivotal axis of the disc is about 1 mm. With this slight offset between pivotal axis and disc mid-plane, when the disc is tilted it is displaced relative to the slideway of the sliding blocks only far enough that on one side of the disc the degree to which the slideway overlaps the outer edge of the disc is reduced. Although it is theoretically possible to shift the disc so far that the slideway is confined entirely to the disc in the region of one dead-centre point of the piston, the invention is not intended to produce this effect; as the distance by which the pivotal axis is offset from the mid-plane of the disc increases, the centre of mass of the disc also moves away from the long axis of the drive shaft. The value given above, if the geometric relationships are as described, represents a good

Figure 1 shows an axial piston compressor according to the state of the art. It contains a housing 10 within which a drive shaft 12 is rotatably mounted. To the drive shaft 12 there is attached a swash plate 14, so that it cannot rotate on the

on the swash plate 14. The radius of this slideway corresponds to the distance between the centre of the ball-and-socket joint on each cylinder defined by the receptacles 26 and the long axis L. Because in the exemplary embodiment shown here the centre of each ball-and-socket joint coincides with the long axis Z of each cylinder 16, the radius of the slideway corresponds to the distance between the long axis Z and the long axis L. In contrast, when the swash plate is pivoted out of its position perpendicular to the drive shaft 12, the result is an elliptical slideway. The reason is that at the two dead-centre points of the piston, which are shown in Figure 1, each flat surface is at a greater distance from the pivotal point C of the swash plate 14 than when it is in the intermediate positions, 90° away from the dead-centre points.

To save space, the outside diameter A of the swash plate 14 is made such that in its untilted position the swash plate projects only slightly beyond the radially outer side of the sliding blocks 20; therefore, because when the swash plate 14 is tilted, its outside diameter appears to be shortened to the value A', the slideways of the sliding blocks 20 are no longer completely on the swash plate. Hence the flat surface 22 of the sliding block is no longer completely in contact with the swash plate 14. The amount by which the flat surface 22 projects beyond the outer edge of the swash plate 14 is indicated in the figures by "a". Figure 4 shows the situation at the moment when the piston passes through the upper and the lower dead-centre point with the swash plate 14 tilted at the angle α . In Figure 5 is a projection of a sliding block 20 and the swash plate 14 onto a plane perpendicular to the long axis L of the drive shaft 12 at the moment of passage through a dead-centre point of the piston. It is clear that the sliding block 20 extends beyond the periphery of the disc 14 by the distance a. Given a distance of 30 mm between the long axis Z of the piston and the long axis L of the drive shaft 12, an 8-mm diameter of the flat surface 22 of the sliding blocks 20, and a maximal tilt angle α of 18° , the geometric relationships are such that the overlap

From this description of the changing force acting on the piston, in connection with the geometric relationships, it will be evident that the proportion of the flat surfaces 22 that is available for force transfer is minimal in the region of the lower dead-centre point, i.e. in the region of the transition from suction stroke to compression stroke. However, here the increase in surface pressure brought about by the fact that

only part of the flat surface 22 makes contact with the swash plate 14 is not critical, because in this region relatively small forces are being transferred. In the region of the top-dead-centre point the flat surfaces of the sliding blocks project just as far beyond the edge of the swash plate 14, but it is here that the strongest forces must be transferred between the swash plate 14 and the sliding blocks 20; hence there is a critical increase in surface pressure between the flat surface 22 in this position and the corresponding part of the swash plate 14. This surface pressure can become so great as to cause severe abrasion between the swash plate 14 and the flat surface 22 of the sliding block 20.

The increased surface pressure just described, between the sliding blocks 20 and the swash plate 14 in the top-dead-centre region of the associated piston, can be reduced or eliminated by the configuration in accordance with the invention, which is shown schematically in Figure 6. In contrast to the configuration known in the state of the art, here the pivotal axis C is offset from the mid-plane of the swash plate 14 by the dimension V. The offset V is such that the pivotal axis C is situated on the side of the swash plate 14 that faces the pistons (not shown in Fig. 6) that it drives. Because of the offset V, when the swash plate 14 is pivoted it makes a translational as well as a rotational movement. As a result, the outer edge of the swash plate 14 is eccentrically disposed with respect to its position at the dead-centre points of the pistons. By this means, the slideway 20 of the sliding blocks is again entirely confined to the surface of the swash plate 14 in the top-dead-centre region of the associated piston; the overlap distance a is equal to zero. Hence the entire area of the flat surface 22 is made available for transferring force. Set against this benefit is the fact that the overlap of the sliding block is doubled in the part of the slideway corresponding to the low-pressure point of the piston movement; however, the resulting increase in surface pressure is

- 13 -

Claims

1. Axial piston compressor with a drive shaft (12) for a disc
(14) that is mounted on the drive shaft in such a way that
5 it can be tilted relative to the drive shaft about a
pivotal axis (C), and at least one piston (18), wherein the
pivotal axis (C) of the disc (14) is disposed eccentrically
with respect to the mid-plane of the disc,
characterized in that the piston (18) is provided with at
10 least two sliding blocks (20) that move along the disc (14)
on a slideway, arranged such that the piston (18) encloses
the sliding blocks (20) in a C-shaped structure, and that
the position of the pivotal axis (C) relative to the mid-
plane of the disc is on the side that faces the the piston
15 (18), so that the disc (14) can be moved relative to the
sliding blocks (20) in such a way that the slideway of the
sliding blocks projects beyond the the edge of the disc
only slightly or not at all.
2. Axial piston compressor according to Claim 1,
20 characterized in that the disc is a swash plate (14), which
can be set into rotation by the drive shaft (12) and can be
adjusted to various tilt angles (α) with respect to the
drive shaft.
3. Axial piston compressor according to Claim 1 or 2,
25 characterized in that the disc is a wobble plate that is
rotatably mounted on a swash plate and is set at a tilt
angle with respect to the drive shaft that corresponds to
the angle of the swash plate.

4. Axial piston compressor according to one of the preceding claims,

characterized in that, given a distance of 30 mm between the long axis (L) of the drive shaft and the long axis (Z) of the piston, an 8-mm diameter of the flat surface (22) of the sliding blocks (20), which is apposed to the slideway, and a maximal tilt angle (α) of 18° between the long axis of the drive shaft and the central axis of the disc, the distance between the mid-plane of the disc and the pivotal axis of the disc (14) is no greater than about 1 mm.

ABSTRACT

Axial piston compressor

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In an axial piston compressor with a drive shaft (12), a disc (14) that is mounted on the drive shaft in such a way that it can be tilted relative to the drive shaft about a pivotal axis (C), and at least one piston (18) provided with sliding blocks (20) that move along the disc (14) on a slideway, the objective is to reduce the maximal surface pressure acting between the disc (14) and the sliding blocks (20). For this purpose it is provided that the pivotal axis (C) of the disc (14) is disposed eccentrically with respect to the mid-plane of the disc.

10

15

Figure 6

VERSION WITH MARKINGS TO SHOW CHANGES MADE

Attorney Docket No. 825-164

In the Specification:

Please add the following paragraph at page 1, between the title and the first line of text as follows:

CROSS REFERENCE TO RELATED APPLICATION

The present application is the U.S. national stage application of International Application PCT/EP00/09705, filed October 4, 2000, which international application was published on April 12, 2001 as International Publication WO 01/25635 A1. The International Application claims priority of German Patent Application 199 47 677.2, filed October 4, 1999.

In the Claims:

Claim 3 has been amended as follows:

3. (amended) Axial piston compressor according to Claim 1 ~~or 2~~, characterized in that the disc is a wobble plate that is rotatably mounted on a swash plate and is set at a tilt angle with respect to the drive shaft that corresponds to the angle of the swash plate.

Claim 4 has been amended as follows:

4. (amended) Axial piston compressor according to ~~one of the preceding claims~~ claim 1,

characterized in that, given a distance of 30 mm between the long axis (L) of the drive shaft and the long axis (Z) of the piston, an 8-mm diameter of the flat surface (22) of the sliding blocks (20), which is ~~apposed~~ opposed to the slideway, and a maximal tilt angle (α) of 18° between the long axis of the drive shaft and the central axis of the disc, the distance between the mid-plane of the disc and the pivotal axis of the disc (14) is no greater than about 1 mm.

(12) NACH DEM VERTRAG ÜBER DIE INTERNATIONALE ZUSAMMENARBEIT AUF DEM GEBIET DES
PATENTWESENS (PCT) VERÖFFENTLICHTE INTERNATIONALE ANMELDUNG(19) Weltorganisation für geistiges Eigentum
Internationales Büro(43) Internationales Veröffentlichungsdatum
12. April 2001 (12.04.2001)

PCT

(10) Internationale Veröffentlichungsnummer
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4. Oktober 2000 (04.10.2000)

(25) Einreichungssprache: Deutsch

(26) Veröffentlichungssprache: Deutsch

(30) Angaben zur Priorität:
199 47 677.2 4. Oktober 1999 (04.10.1999) DE(71) Anmelder (für alle Bestimmungsstaaten mit Ausnahme von
US): ZEXEL GMBH [DE/DE]; Zeppelinstrasse 5, 64331
Weiterstadt (DE).

(72) Erfinder; und

(75) Erfinder/Anmelder (nur für US): SCHWARZKOPF, Ot-
fried [DE/DE]; Kniebissstrasse 18/1, 71106 Magstadt (DE).(74) Anwälte: POPP, Eugen usw.; Meissner, Bolte & Partner,
Postfach 86 06 24, 81633 München (DE).(81) Bestimmungsstaaten (national): AE, AG, AL, AM, AT,
AU, AZ, BA, BB, BG, BR, BY, BZ, CA, CH, CN, CR, CU,
CZ, DK, DM, DZ, EE, ES, FI, GB, GD, GE, GH, GM, HR,
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LS, LT, LU, LV, MA, MD, MG, MK, MN, MW, MX, MZ,
NO, NZ, PL, PT, RO, RU, SD, SE, SG, SI, SK, SL, TJ, TM,
TR, TT, TZ, UA, UG, US, UZ, VN, YU, ZA, ZW.(84) Bestimmungsstaaten (regional): ARIPO-Patent (GH,
GM, KE, LS, MW, MZ, SD, SL, SZ, TZ, UG, ZW), eura-
sisches Patent (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM),
europäisches Patent (AT, BE, CH, CY, DE, DK, ES, FI,
FR, GB, GR, IE, IT, LU, MC, NL, PT, SE), OAPI-Patent
(BF, BJ, CF, CG, CI, CM, GA, GN, GW, ML, MR, NE,
SN, TD, TG).

Veröffentlicht:

— Mit internationalem Recherchenbericht.

Zur Erklärung der Zweibuchstaben-Codes, und der anderen
Abkürzungen wird auf die Erklärungen ("Guidance Notes on
Codes and Abbreviations") am Anfang jeder regulären Ausgabe
der PCT-Gazette verwiesen.

(54) Title: AXIAL PISTON DISPLACEMENT COMPRESSOR

(54) Bezeichnung: AXIALKOLBENVERDICHTER

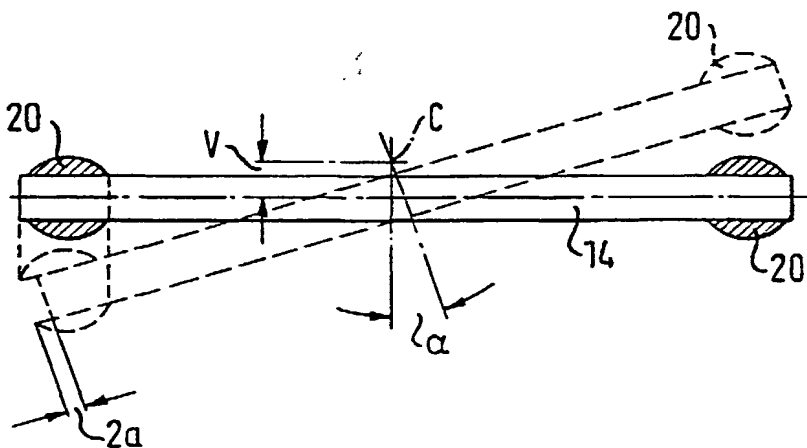
(57) Abstract: The invention relates
to an axial piston displacement
compressor, comprising a drive shaft
(12), a disc (14) which is mounted
on the drive shaft in such a way, that
it can be pivoted about a pivotal axis
(C) and at least one piston (18) which
is provided with sliding blocks (20)
that slide on a runway on the disc
(14). The invention aims to reduce the
maximum effective surface pressure
between the disc (14) and the sliding
blocks (20). To this end, the pivotal
axis (C) of the disc (14) is located
eccentrically in relation to the central
plane of the disc.(57) Zusammenfassung: Bei einem
Axialkolbenverdichter mit einerAntriebswelle (12), einer Scheibe (14), die auf der Antriebswelle so gelagert ist, dass sie relativ zur Antriebswelle um eine
Schwenkachse (C) verschwenkt werden kann, und mindestens einem Kolben (18), der mit Gleitsteinen (20) versehen ist, die auf
einer Laufbahn auf der Scheibe (14) gleiten, soll die maximal zwischen der Scheibe (14) und den Gleitsteinen (20) wirkende
Flächenpressung verringert werden. Zu diesem Zweck ist vorgesehen, dass die Schwenkachse (C) der Scheibe (14) exzentrisch
bezüglich der Mittelebene der Scheibe angeordnet ist.

FIG. 1
(Prior Art)

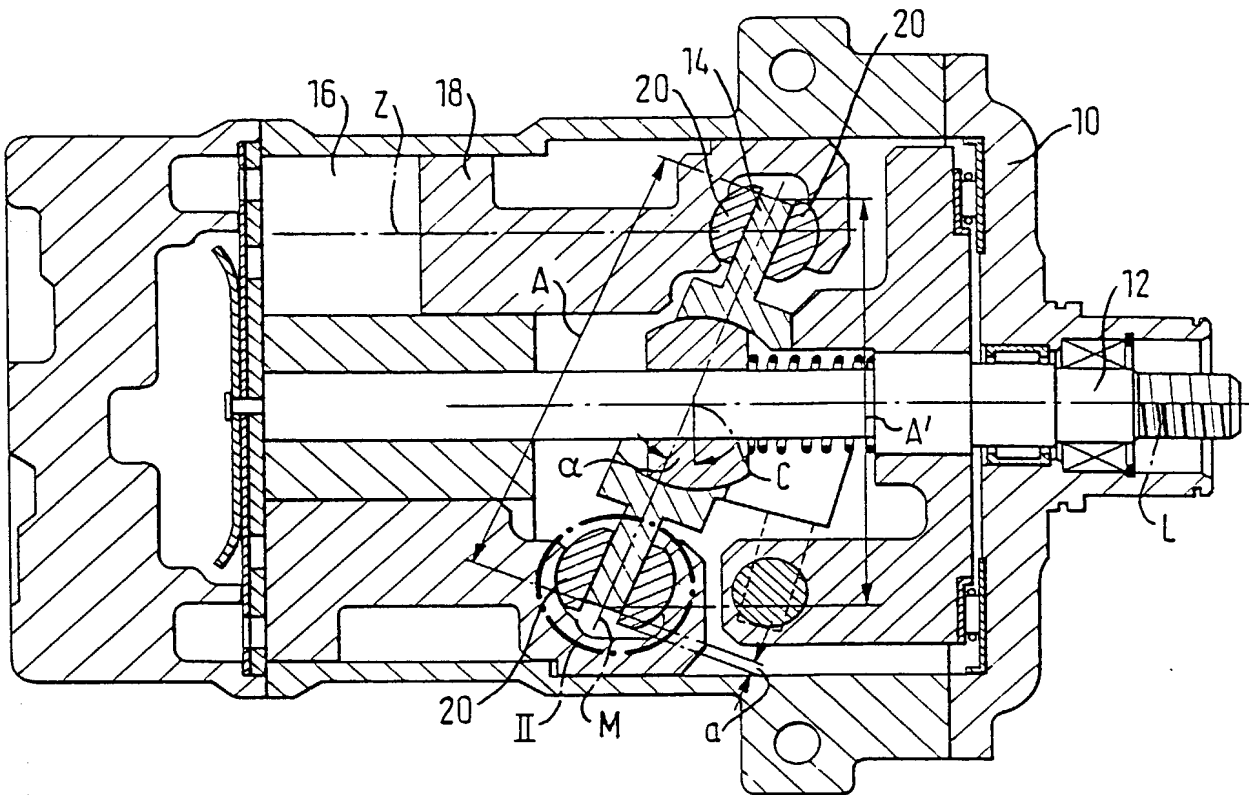
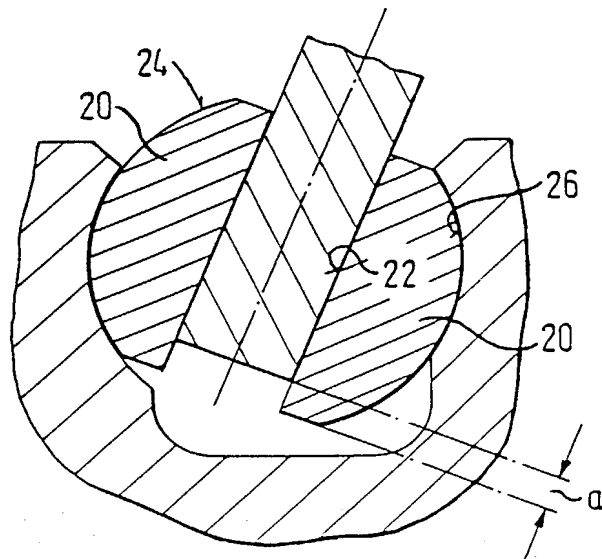
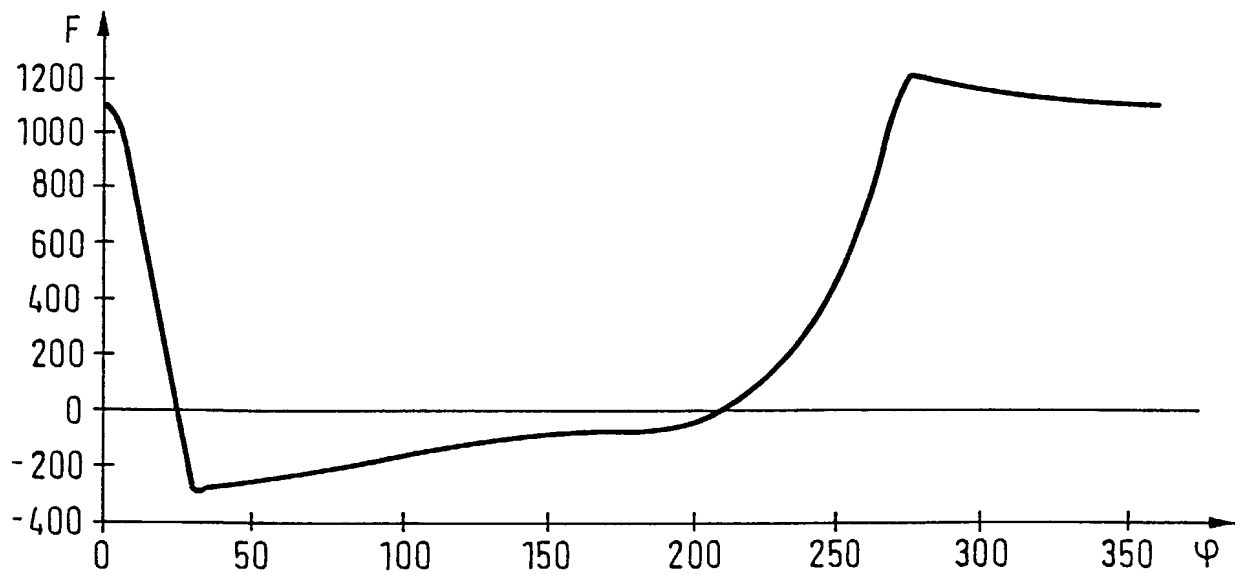


FIG. 2



2/3

FIG. 3



3/3

FIG. 4

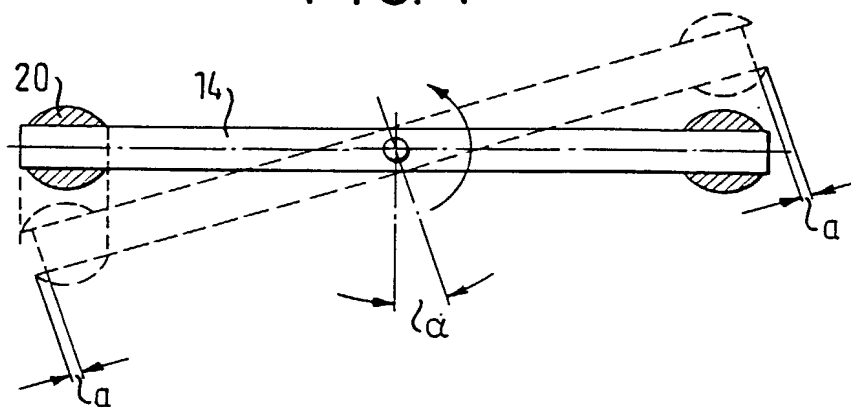


FIG. 6

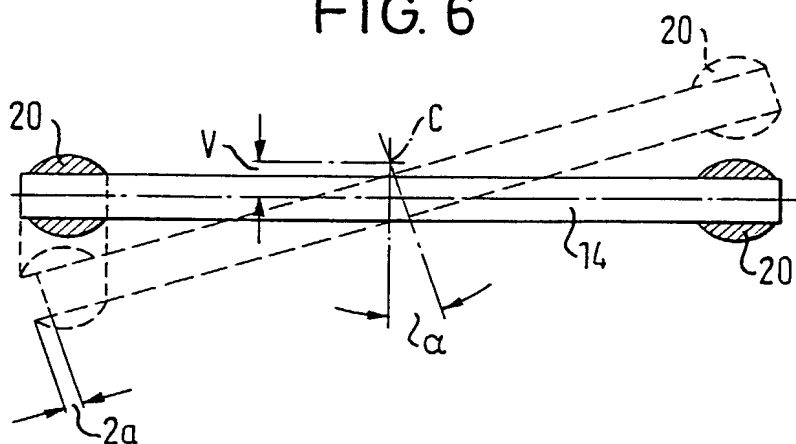
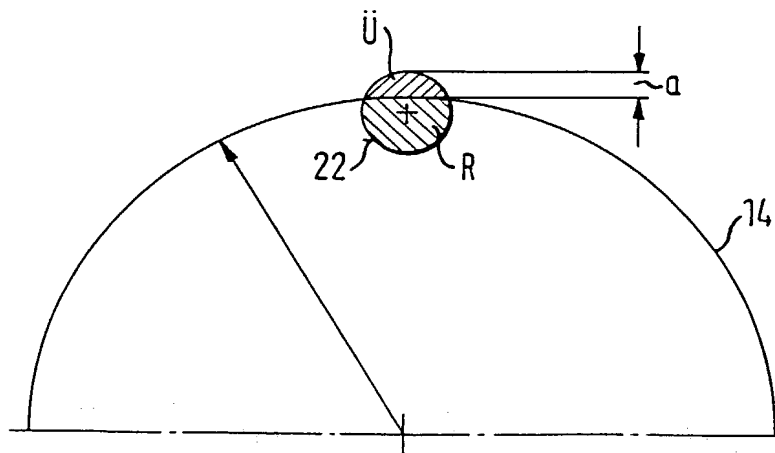


FIG. 5



17/12EX-021-PLIUS

10/089552 08 JUL 2002

Type a plus sign (+) inside this box [+]

Approved for use through 9/30/00

Patent and Trademark Office: U.S. DEPARTMENT OF COMMERCE

PTO/SB/01 (8/96)	<table border="1" style="width:100%; border-collapse: collapse;"> <tr> <td style="width:50%;">Attorney Docket Number</td> <td>825-164</td> </tr> <tr> <td>First Named Inventor</td> <td>Otfried Schwarzkopf</td> </tr> <tr> <td colspan="2" style="text-align: center;">COMPLETE IF KNOWN</td> </tr> <tr> <td>Application Number</td> <td></td> </tr> <tr> <td>Filing Date</td> <td></td> </tr> <tr> <td>Group Art Unit</td> <td></td> </tr> <tr> <td>Examiner Name</td> <td></td> </tr> </table>	Attorney Docket Number	825-164	First Named Inventor	Otfried Schwarzkopf	COMPLETE IF KNOWN		Application Number		Filing Date		Group Art Unit		Examiner Name	
Attorney Docket Number	825-164														
First Named Inventor	Otfried Schwarzkopf														
COMPLETE IF KNOWN															
Application Number															
Filing Date															
Group Art Unit															
Examiner Name															

DECLARATION

Declaration OR Declaration

☐ Submitted with ☒ Submitted after

Initial Filing Initial Filing

As a below named inventor, I hereby declare that:

My residence, post office address, and citizenship are as stated below next to my name.

I believe I am the original, first and sole inventor (if only one name is listed below) or an original, first and joint inventor (if plural names are listed below) of the subject matter which is claimed and for which a patent is sought on the invention entitled:

AXIAL PISTON COMPRESSOR

(Title of the Invention)

the specification of which

☐ is attached hereto

OR

☒ was filed on (MM/DD/YYYY) October 4, 2000 as United States Application Number or PCT

International Number PCT/EP00/09705 and was amended on (MM/DD/YYYY) July 7, 2001 (if applicable).

I hereby state that I have reviewed and understand the contents of the above identified specification, including the claims, as amended by any amendment specifically referred to above.

I acknowledge the duty to disclose information which is material to patentability as defined in Title 37 Code of Federal Regulations, §1.56.

I hereby claim foreign priority benefits under Title 35, United States Code §119(a)-(d) or §365(b) of any foreign application(s) for patent or inventor's certificate, or §365(a) of any PCT international application which designed at least one country other than the United States of America, listed below and have also identified below, by checking the box, any foreign application for patent or inventor's certificate, or of any PCT international application having a filing date before that of the application on which priority is claimed.

Prior Foreign Application Number(s)	Country	Foreign Filing Date (MM/DD/YYYY)	Priority Not Claimed	Copy Attached?	
				YES	NO
199 47 677.2	Germany	10/04/99	<input type="checkbox"/>	<input type="checkbox"/>	<input checked="" type="checkbox"/>
			<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>
			<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>
			<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>
			<input type="checkbox"/>	<input type="checkbox"/>	<input type="checkbox"/>

☐ Additional foreign application numbers are listed on a supplemental priority sheet attached hereto:

I hereby claim the benefit under Title 35, United States Code §119(e) of any United States provisional application(s) listed below.

Application Number(s)	Filing Date (MM/DD/YYYY)	Additional provisional
		<input type="checkbox"/> Application numbers are listed on a supplemental priority sheet attached hereto.

Type a plus sign (+) inside this box [+]

DECLARATION

I hereby claim the benefit under Title 35, United States Code §120 of any United States application(s), or §365© of any PCT international application designated the United States of America, listed below and, insofar as the subject matter of each of the claims of this application is not disclosed in the prior United States of PCT International application in the manner provided by the first paragraph of Title 35, United States Code §112. I acknowledge the duty to disclose information which is material to patentability as defined in Title 37, Code of Federal Regulations §1.56 which became available between the filing date of the prior application and the national or PCT international filing date of this application.

U.S. Parent Application Number	PCT Parent Number	Parent Filing Date (MM/DD/YYYY)	Parent Patent Number (if applicable)

☐ Additional U.S. or PCT international application numbers are listed on a supplemental priority sheet attached hereto.

As a named inventor, I hereby appoint the following attorney(s) and/or agent(s) to prosecute this application and to transact all business in the Patent and Trademark Office connected therewith:

Name	Registration Number	Name	Registration Number
Daniel D. Fetterley	20,323	Joseph D. Kuborn	40,689
George H. Solveson	25,927	William L. Falk	27,709
Gary A. Essmann	29,376	Jeffrey S. Sokol	35,686
Thomas M. Wozny	28,922		
Michael E. Taken	28,120		
Joseph J. Jochman, Jr.	25,058		

☐ Additional attorney(s) and/or agent(s) named on a supplemental sheet attached hereto.

☒ Please direct all correspondence to: Name Gary A. Essmann

Address	Andrus, Sceales, Starke & Sawall, LLP		
Address	100 East Wisconsin Avenue, Suite 1100		
City	Milwaukee	State	Wisconsin
Country	United States	Telephone	(414) 271-7590
		Fax	(414) 271-5770

I hereby declare that all statements made herein of my own knowledge are true and that all statements made on information and belief are believed to be true; and further that these statements were made with the knowledge that willful false statements and the like so made are punishable by fine or imprisonment, or both, under §1001 of Title 18 of the United States Code and that such willful false statements may jeopardize the validity of the application or any patent issued thereon.

Name of Sole or First Inventor: ☐ A petition has been filed for this unsigned inventor

Given Name (first and middle [if any])	Family Name or Surname
OTFRIED	SCHWARZKOPF

Inventor's Signature	<i>Offert Silevassan</i>	Date	29/05/02
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RESIDENCE: City	Magstadt	State		Country	Germany	Citizenship	German
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